

Lecture 10, Feb 8, 2024

Second Order System Response (Continued)

- Consider the step response

$$\begin{aligned}
 - y_s(t) &= \mathcal{L}^{-1} \left\{ \frac{H(s)}{s} \right\} \\
 &= \mathcal{L}^{-1} \left\{ \frac{\sigma^2 + \omega_d^2}{(s + \sigma)^2 + \omega_d^2} s \right\} \\
 &= \mathcal{L}^{-1} \left\{ \frac{1}{s} - \frac{s + 2\sigma}{(s + \sigma)^2 + \omega_d^2} \right\} \\
 &= \mathcal{L}^{-1} \left\{ \frac{1}{s} - \frac{s + \sigma}{(s + \sigma)^2 + \omega_d^2} - \frac{\sigma}{\omega_d} \frac{\omega_d}{(s + \sigma)^2 + \omega_d^2} \right\} \\
 &= 1 - e^{-\sigma t} \left(\cos(\omega_d t) + \frac{\sigma}{\omega_d} \sin(\omega_d t) \right) \\
 &= 1 - e^{-\sigma t} \frac{\omega_n}{\omega_d} \cos(\omega_d t - \theta)
 \end{aligned}$$

* Where $\theta = \tan^{-1} \left(\frac{\omega_d}{\sigma} \right) = \tan^{-1} \left(\frac{\omega_d}{\zeta \omega_n} \right)$

- For an overdamped system, the two separate poles lie on the real axis, and with decreasing ζ the poles move together until they overlap, and then move radially into the imaginary axis
- The system starts with no oscillation but a slow response to faster responses but oscillations begin; when $\zeta = 0$ the poles are purely imaginary, at which point the response is purely oscillatory and no decay occurs
 - When $\zeta = 1$, the poles overlap, and we get critical damping, which is the fastest possible system response without oscillation

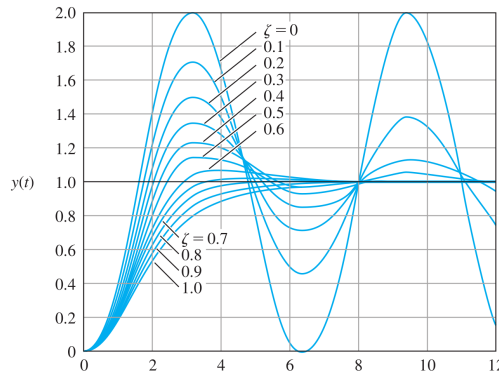


Figure 1: Step response of an underdamped second-order system.

- For the second order system, we can characterize it using the following (for a unit step input):
 - DC gain y_{ss} : the steady-state value of the system output
 - Peak time t_p : time to reach the maximum overshoot/undershoot point
 - Overshoot M_p : the max amount the output overshoots y_{ss} , divided by the steady state value (usually as a percentage)
 - Rise time t_r : the time the system takes to rise from 10% to 90% of y_{ss}
 - Settling time t_s : the time the system takes to reach, and stay within, 1% of y_{ss} (2% in some texts)
- DC gain: $y_{ss} = \lim_{s \rightarrow 0} sY(s) = \lim_{s \rightarrow 0} s \frac{1}{s} H(s) = \frac{\omega_n^2}{\omega_n^2} = 1$
 - The DC gain here is 1 because when we derived the system, we multiplied u by k
 - Without this scaling the DC gain would be k instead

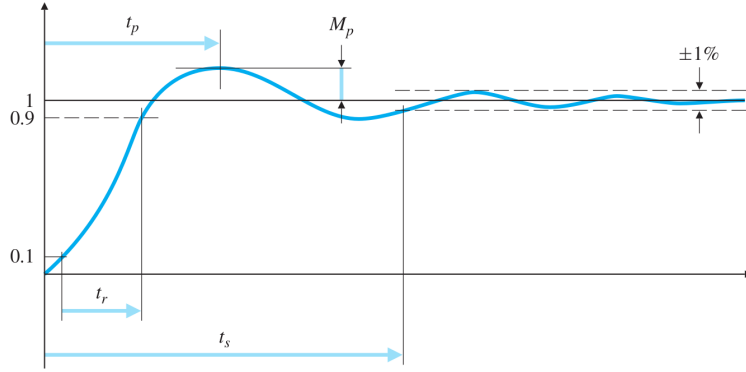


Figure 2: Illustration of the characteristics of second-order system response.

- Peak time:
 - Take derivative: $\dot{y}_s(t) = \mathcal{L}^{-1} \{sY_s(s)\} = \mathcal{L}^{-1} \left\{ s \frac{1}{s} H(s) \right\} = y_i(t)$
 - * Note the derivative of the step response is just the impulse response
 - Therefore $y_i(t) = \frac{\omega_n}{\sqrt{1-\zeta^2}} e^{-\zeta\omega_n t} \sin(\omega_n \sqrt{1-\zeta^2} t)$
 - $\dot{y}_s(t) = y_i(t) = 0 \implies \omega_n \sqrt{1-\zeta^2} t = n\pi \implies t = \frac{n\pi}{\omega_n \sqrt{1-\zeta^2}}$
 - The first peak occurs at $t_p = \frac{\pi}{\omega_n \sqrt{1-\zeta^2}} = \frac{\pi}{\omega_d}$
 - As we reduce the damping,
- Overshoot:
 - Substitute t_p into the step response to get the peak of the response
 - $y_s(t_p) = 1 - \frac{e^{-\frac{\pi\zeta}{\sqrt{1-\zeta^2}}}}{\sqrt{1-\zeta^2}} \cos(\pi - \theta) = 1 + \frac{e^{-\frac{\pi\zeta}{\sqrt{1-\zeta^2}}}}{\sqrt{1-\zeta^2}} \cos \theta$
 - We know $\cos \theta = \sqrt{1-\zeta^2}$ so this simplifies to $1 + e^{-\frac{\pi\zeta}{\sqrt{1-\zeta^2}}}$
 - The overshoot is therefore $M_p = e^{-\frac{\pi\zeta}{\sqrt{1-\zeta^2}}}$ (or times 100 for percentage)
 - Notice that this depends only on ζ
 - * Usually we're interested in two values: $\zeta = \frac{1}{2}$ which gives 16% overshoot, and $\zeta = 0.7$ which gives 5% overshoot
 - The percent overshoot decreases with ζ , but $\omega_n t_p$ increases with ζ
- Rule of thumb: the response of the second-order underdamped systems (with no finite zeroes) with different damping ratios rises roughly with the same pace
 - Typically we related t_r to only ω_n instead of also ζ , as an approximation
 - For $\zeta = 0.5$, we can approximate $t_r \approx \frac{1.8}{\omega_n}$
 - We typically choose ζ between 0.5 and 0.7 for a balance between overshoot and rise time
- For settling time we can approximate the deviation of the response by the exponential only
 - Therefore $e^{-\zeta\omega_n t_s} \approx 0.01 \implies t_s \approx \frac{4.6}{\zeta\omega_n} = \frac{4.6}{\sigma}$

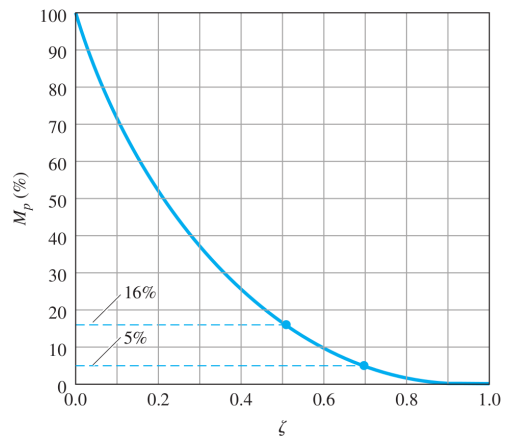


Figure 3: Overshoot as a function of damping ratio.