

# Lecture 22, Nov 30, 2023

## Modal Analysis

- Consider first the unforced equation of motion,  $M\ddot{\mathbf{q}} + \mathbf{K}\mathbf{q} = \mathbf{0}$
- Substituting in the test solution  $\mathbf{q}_0 e^{\lambda t}$  gives us  $(\lambda^2 \mathbf{M} + \mathbf{K})\mathbf{q}_0 = \mathbf{0}$ 
  - Clearly, this is satisfied for the trivial solution  $\mathbf{q}_0 = \mathbf{0}$ , but we want some non-quiescent solution
  - For  $\mathbf{q}_0$  to be nonzero, we need  $\det(\lambda^2 \mathbf{M} + \mathbf{K}) = 0$ ; this resembles an eigenproblem where instead of identity we have  $\mathbf{M}$
  - There will be multiple such  $\lambda$  and  $\mathbf{q}_0$
- $\lambda_\alpha^2$  are then the eigenvalues and  $\mathbf{q}_\alpha$  the eigenvectors;  $\det(\lambda^2 \mathbf{M} + \mathbf{K}) = 0$  is the characteristic equation or eigenequation
  - In general the eigenequation gives us an  $n$ -th order polynomial in  $\lambda^2$
  - Consider multiplying both sides by the Hermitian of  $\mathbf{q}_\alpha$ :  $\lambda_\alpha^2 \mathbf{q}_\alpha^H \mathbf{M} \mathbf{q}_\alpha + \mathbf{q}_\alpha^H \mathbf{K} \mathbf{q}_\alpha = 0$ 
    - \*  $\mathbf{M}$  is real and symmetric, so  $\mathbf{q}_\alpha^H \mathbf{M} \mathbf{q}_\alpha$  is real; furthermore its positive-definiteness means this is also always greater than 0
    - \*  $\mathbf{K}$  is real and symmetric, so  $\mathbf{q}_\alpha^H \mathbf{K} \mathbf{q}_\alpha$  is also real
    - \*  $\lambda_\alpha^2 = -\frac{\mathbf{q}_\alpha^H \mathbf{K} \mathbf{q}_\alpha}{\mathbf{q}_\alpha^H \mathbf{M} \mathbf{q}_\alpha}$  so indeed  $\lambda$  is real
    - \* Further,  $\mathbf{K} > 0 \implies \lambda_\alpha^2 < 0$ , or  $\lambda_\alpha = \pm j\omega_\alpha$
    - \* By extension, the  $\mathbf{q}_\alpha$  are real
- Do  $\mathbf{q}_\alpha$  form a basis?
  - Consider  $\lambda_\alpha^2 \mathbf{q}_\beta^T \mathbf{M} \mathbf{q}_\alpha + \mathbf{q}_\beta^T \mathbf{K} \mathbf{q}_\alpha = 0$  and  $\lambda_\beta^2 \mathbf{q}_\alpha^T \mathbf{M} \mathbf{q}_\beta + \mathbf{q}_\alpha^T \mathbf{K} \mathbf{q}_\beta = 0$
  - Subtracting the two equations gives  $(\lambda_\alpha^2 - \lambda_\beta^2) \mathbf{q}_\alpha^T \mathbf{M} \mathbf{q}_\beta = 0$  (note we can do this since the terms are scalars)
  - Then  $\mathbf{q}_\alpha^T \mathbf{M} \mathbf{q}_\beta = \begin{cases} > 0 & \lambda_\alpha^2 = \lambda_\beta^2 \\ 0 & \lambda_\alpha^2 \neq \lambda_\beta^2 \end{cases}$
  - WLOG normalize the  $\mathbf{q}$  vectors with respect to  $\mathbf{M}$ , then  $\mathbf{q}_\alpha^T \mathbf{M} \mathbf{q}_\beta = \delta_{\alpha\beta} \implies \mathbf{Q}^T \mathbf{M} \mathbf{Q} = \mathbf{1}$  where  $\mathbf{Q} = [\mathbf{q}_1 \ \cdots \ \mathbf{q}_m]$ 
    - \* Note we might get repeated eigenvalues, but we can always diagonalize due to the symmetry of  $\mathbf{M}$
  - Plugging back into the first equation,  $\lambda_\alpha^2 \delta_{\alpha\beta} + \mathbf{q}_\alpha^T \mathbf{K} \mathbf{q}_\beta^T \implies \mathbf{q}_\alpha^T \mathbf{K} \mathbf{q}_\beta^T = \begin{cases} -\lambda_\alpha^2 & \alpha = \beta \\ 0 & \alpha \neq \beta \end{cases}$
  - So we can also write  $\mathbf{Q}^T \mathbf{K} \mathbf{Q} = -\Lambda^2$
- Let  $\mathbf{q}(t) = \sum_{\beta=1}^n \mathbf{q}_\beta \eta_\beta(t)$ 
  - $\sum_{\beta=1}^n \mathbf{M} \mathbf{q}_\beta \ddot{\eta}_\beta + \sum_{\beta=1}^n \mathbf{K} \mathbf{q}_\beta \eta_\beta = \mathbf{f}$
  - $\implies \sum_{\beta=1}^n \mathbf{q}_\alpha^T \mathbf{M} \mathbf{q}_\beta \ddot{\eta}_\beta + \sum_{\beta=1}^n \mathbf{q}_\alpha^T \mathbf{K} \mathbf{q}_\beta \eta_\beta = \mathbf{q}_\alpha^T \mathbf{f}$
  - $\implies \sum_{\beta=1}^n \delta_{\alpha\beta} \ddot{\eta}_\beta + \sum_{\beta=1}^n -\lambda_\alpha^2 \eta_\beta = \mathbf{q}_\alpha^T \mathbf{f}$
  - $\implies \ddot{\eta}_\alpha - \lambda_\alpha^2 \eta_\alpha = f_\alpha$
  - We have uncoupled the system of differential equations
  - $\mathbf{q}_\beta$  are the *mode shapes*, and  $\eta_\beta(t)$  are the *modal coordinates*;  $\mathbf{q}_\alpha \eta_\alpha$  is a *mode* of vibration
  - Note that if  $\mathbf{K}$  were positive definite, we would get all negative  $\lambda_\alpha^2$ , giving oscillatory motion; then  $\omega_\alpha^2 = -\lambda_\alpha^2$  are the *natural frequencies* of vibration

## Double Pendulum Revisited

- Recall:  $\mathbf{M} = ml^2 \begin{bmatrix} 2 & 1 \\ 1 & 1 \end{bmatrix}$ ,  $\mathbf{K} = mgl \begin{bmatrix} 2 & 0 \\ 0 & 1 \end{bmatrix}$
- Clearly  $\mathbf{K}$  is positive definite here, so let's write  $\lambda^2$  as  $-\omega^2$
- We want to solve  $\det(-\omega_\alpha^2 \mathbf{M} + \mathbf{K}) = 0$
- Let  $\mu_\alpha^2 = -\omega_\alpha^2 \frac{l}{g}$ , so that  $\det \begin{bmatrix} -2\mu_\alpha^2 + 2 & -\mu_\alpha^2 \\ -\mu_\alpha^2 & -\mu_\alpha^2 + 1 \end{bmatrix} = 0$
- Expanded:  $\mu^3 - 4\mu^2 + 2 = 0 \implies \mu^2 = 2 \pm \sqrt{2}$
- Therefore the modal frequencies are  $\omega_1 = \sqrt{(2 - \sqrt{2})\frac{g}{l}}$ ,  $\omega_2 = \sqrt{(2 + \sqrt{2})\frac{g}{l}}$
- Solve the eigenequation to get  $\frac{\theta_{1,2}}{\theta_{1,1}} = \sqrt{2}$  and  $\frac{\theta_{2,2}}{\theta_{2,1}} = -\sqrt{2}$
- For each of the modes, at any time, the ratio of the coordinates remains the same
- Notice that in the second mode, we have a *node* – a point that does not move
  - In general, for an  $n$  degree of freedom system, we will have  $n$  modes; mode  $n$  will have  $n - 1$  nodes

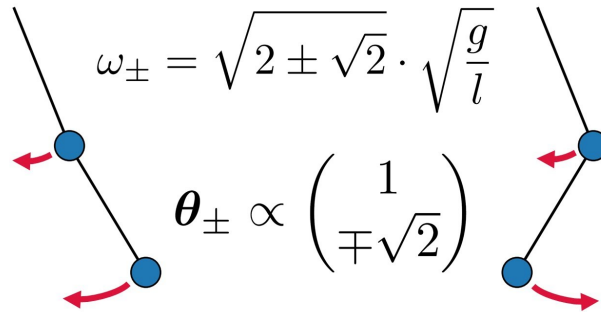


Figure 1: Vibrational modes of the double pendulum.