

# Lecture 18, Nov 16, 2023

## Dynamics of Rigid Bodies

### Momentum of Rigid Bodies

- Consider a rigid body  $R$  and some inertial reference point  $O_{\mathcal{J}}$ 
  - Each differential mass element  $dm$  has momentum  $d\underline{p} = \underline{r}' dm$
  - Therefore the overall momentum is  $\underline{p} = \int_R \underline{r}' dm$
- Consider some reference point  $O$  fixed to the body, and let  $\underline{\rho}$  be the position of a mass element relative to  $O$ 
  - $\underline{r}' = \underline{v}_O + \underline{\rho}' = \underline{v}_O + \underline{\rho}'^{\circ} + \underline{\omega} \times \underline{\rho}$
  - But  $\underline{\rho}$  is fixed as seen in a body-relative frame, so  $\underline{\rho}'^{\circ} = 0$  (unless the body is deformable)
  - $\underline{p} = \int_R (\underline{v}_O + \underline{\omega} \times \underline{\rho}) dm = \int_R \underline{v}_O dm - \int_R \underline{\rho} \times \underline{\omega} dm = m\underline{v}_O - \left( \int_R \underline{\rho} dm \right) \times \underline{\omega}$
- Let  $\underline{c}_O = \int_R \underline{\rho} dm$  be the *first moment of mass* (aka *first moment of inertia*), then  $\underline{p} = m\underline{v}_O - \underline{c}_O \times \underline{\omega}$ 
  - Note that  $\underline{c}_O$  has a subscript since it is computed with respect to  $O$
  - Expressed in a body frame  $\underline{\mathcal{F}}_b$ ,  $\underline{p} = m\underline{v}_O - \underline{c}_O^{\times} \underline{\omega}$ 
    - \* Note  $\underline{c}_O$  is a constant in  $\underline{\mathcal{F}}_b$
- For angular momentum,  $d\underline{h}_O = \underline{\rho} \times d\underline{p} = \underline{\rho} \times (\underline{v}_O - \underline{\rho} \times \underline{\omega}) dm \implies \underline{h}_O = \underline{c}_O \times \underline{v}_O - \int_R \underline{\rho} \times (\underline{\rho} \times \underline{\omega}) dm$ 
  - In  $\underline{\mathcal{F}}_b$ ,  $\underline{h}_O = \underline{c}_O^{\times} \underline{v}_O - \int_R \underline{\rho}^{\times} \underline{\rho}^{\times} \underline{\omega} dm$ 

$$= \underline{c}_O^{\times} \underline{v}_O + \left( - \int_R \underline{\rho}^{\times} \underline{\rho}^{\times} dm \right) \underline{\omega}$$

$$= \underline{c}_O^{\times} \underline{v}_O + \underline{J}_O \underline{\omega}$$
  - $\underline{J}_O = - \int_R \underline{\rho}^{\times} \underline{\rho}^{\times} dm$  is the *second moment of mass*, or the *inertia matrix*
    - \* Note  $\underline{J}_O$  is a *second order tensor*; in vector form,  $\underline{h}_O = \underline{c}_O \times \underline{v}_O + \underline{J}_O \cdot \underline{\omega}$
    - \*  $\underline{J}_O = \int_R (\rho^2 \mathbf{1} - \underline{\rho} \underline{\rho}^T) dm$
  - Let  $\underline{s}$  be any vector, then  $\underline{s}^T \underline{J}_O \underline{s} = -\underline{s}^T \left( \int_R \underline{\rho}^{\times} \underline{\rho}^{\times} dm \right) \underline{s}$ 

$$= - \int_R \underline{s}^T \underline{\rho}^{\times} \underline{\rho}^{\times} \underline{s} dm$$

$$= \int_R (\underline{\rho}^{\times} \underline{s})^T (\underline{\rho}^{\times} \underline{s}) dm$$

$$= \int_R \|\underline{\rho}^{\times} \underline{s}\|^2 dm$$
    - \* There will always be some  $\underline{\rho}$  that is not parallel to  $\underline{s}$  for any 3D body, so this integral is always positive for a nonzero  $\underline{s}$
    - \* Therefore  $\underline{J}_O$  is symmetric positive definite (hence why we include a minus sign in the definition)
- A *second-order tensor*  $\underline{D} = \underline{ab}$  is defined such that  $\underline{D} \cdot \underline{v} = (\underline{ab}) \cdot \underline{v} = \underline{a}(\underline{b} \cdot \underline{v})$
- In matrix form,  $\begin{bmatrix} \underline{p} \\ \underline{h}_O \end{bmatrix} = \begin{bmatrix} m\mathbf{1} & -\underline{c}_O^{\times} \\ \underline{c}_O^{\times} & \underline{J}_O \end{bmatrix} \begin{bmatrix} \underline{v}_O \\ \underline{\omega} \end{bmatrix}$ , where  $\underline{M} = \begin{bmatrix} m\mathbf{1} & -\underline{c}_O^{\times} \\ \underline{c}_O^{\times} & \underline{J}_O \end{bmatrix}$  is the *mass matrix*, which is also symmetric positive definite
- If we choose  $O = \bullet$ , then  $\int_R \underline{r} dm = 0$ , so  $\underline{c} = 0$ 
  - $\underline{p} = m\underline{v}_O$
  - $\underline{h}_{\bullet} = \underline{J}_{\bullet} \underline{\omega} = \underline{I} \underline{\omega}$ , where we denote  $\underline{I}$  as the inertia matrix about the centre of mass
- Consider two inertia matrices  $\underline{J}_A, \underline{J}_B$  relative to points  $A, B$ ; for a differential mass element, denote

position relative to  $A$  by  $\underline{a}$ , position relative to  $B$  by  $\underline{b}$  and the relative position between  $A$  and  $B$  is  $\underline{\rho}^{BA}$

- $\underline{a} = \underline{b} + \underline{\rho}^{BA}$  in a common body frame (draw this out)
- $\underline{J}_A = - \int_R \underline{a}^\times \underline{a}^\times dm$ 

$$= - \int_R (\underline{b} + \underline{\rho}^{BA})^\times (\underline{b} + \underline{\rho}^{BA})^\times dm$$

$$= - \int_R (\underline{b}^\times \underline{b}^\times + \underline{\rho}^{BA \times} \underline{b}^\times + \underline{b}^\times \underline{\rho}^{BA \times} + \underline{\rho}^{BA \times} \underline{\rho}^{BA \times}) dm$$

$$= - \int_R \underline{b}^\times \underline{b}^\times dm - \underline{\rho}^{BA \times} \int_R \underline{b}^\times dm - \int_R \underline{b}^\times dm \underline{\rho}^{BA \times} - \int dm \underline{\rho}^{BA \times} \underline{\rho}^{BA \times}$$

$$= \underline{J}_B - \underline{\rho}^{BA \times} \underline{c}_B^\times - \underline{c}_B^\times \underline{\rho}^{BA \times} - m \underline{\rho}^{BA \times} \underline{\rho}^{BA \times}$$
- This is the *parallel axis theorem* for an inertia matrix

### Theorem

*Parallel Axis Theorem:* Given an inertia matrix  $\underline{J}_B$  around a point  $B$ , and relative position  $\underline{\rho}^{BA}$  from  $A$  to  $B$ , we can find the inertia matrix around  $A$ ,  $\underline{J}_A$  as:

$$\underline{J}_A = \underline{J}_B - \underline{\rho}^{BA \times} \underline{c}_B^\times - \underline{c}_B^\times \underline{\rho}^{BA \times} - m \underline{\rho}^{BA \times} \underline{\rho}^{BA \times}$$

- Consider the same reference point in two frames  $\underline{\mathcal{F}}_a, \underline{\mathcal{F}}_b$ ; denote  $\underline{J}_a, \underline{J}_b$  be the inertia matrix about this point expressed in the two frames

$$\begin{aligned} - \underline{J}_a &= - \int_R \underline{\rho}_a^\times \underline{\rho}_a^\times dm \\ &= - \int_R (\underline{C}_{ab} \underline{\rho}_b)^\times (\underline{C}_{ab} \underline{\rho}_b)^\times dm \\ &= - \int_R (\underline{C}_{ab} \underline{\rho}_b^\times \underline{C}_{ba}) (\underline{C}_{ab} \underline{\rho}_b^\times \underline{C}_{ba}) dm \\ &= \underline{C}_{ab} \left( - \int_R \underline{\rho}_b^\times \underline{\rho}_b^\times \right) \underline{C}_{ba} \\ &= \underline{C}_{ab} \underline{J}_b \underline{C}_{ba} \end{aligned}$$

- This is the *rotational transformation theorem* for an inertia matrix
- Note for a second-order tensor,  $\underline{J} = \underline{\mathcal{F}}_a^T \underline{J}_a \underline{\mathcal{F}}_a \iff \underline{J}_a = \underline{\mathcal{F}}_a \cdot \underline{J} \cdot \underline{\mathcal{F}}_a^T$ , so this identity follows
- \* The result applies for any second-order tensor

### Motion of Rigid Bodies

- To get the equations of motion, we can treat a rigid body like a grammar of particles
- For a grammar of particles,  $\underline{p}^\cdot = \underline{f}$  and  $\underline{h}_O^\cdot + \underline{v}_O \times \underline{p} = \underline{\tau}_O$ 
  - $\underline{p} = m \underline{v}_O - \underline{c}_O^\times \underline{\omega}$
  - $\underline{h}_O = \underline{c}_O^\times \underline{v}_O + \underline{J}_O \underline{\omega}$
  - But to use these, we have to first convert the derivative  $(\cdot)^\cdot$  with respect to inertial frame into a derivative with respect to body frame
    - \*  $\underline{p}^\circ + \underline{\omega} \times \underline{p} = \underline{f}$
    - \*  $\underline{h}_O^\circ + \underline{\omega} \times \underline{h}_O + \underline{v}_O \times \underline{p} = \underline{\tau}_O$
  - In the body frame:
    - \*  $\underline{\dot{p}} + \underline{\omega}^\times \underline{p} = \underline{f}$
    - \*  $\underline{\dot{h}}_O + \underline{\omega}^\times \underline{h}_O + \underline{v}_O^\times \underline{p} = \underline{\tau}_O$
- Therefore the equations of motion for a rigid body are given by, in the general case:
  - $m \underline{\dot{v}}_O - \underline{c}_O^\times \underline{\dot{\omega}} + m \underline{\omega}^\times \underline{v}_O - \underline{\omega}^\times \underline{c}_O^\times \underline{v}_O = \underline{f}$
  - $\underline{c}_O^\times \underline{\dot{v}}_O + \underline{J}_O \underline{\dot{\omega}} - \underline{c}_O^\times \underline{\omega}^\times \underline{v}_O + \underline{\omega}^\times \underline{J}_O \underline{\omega} = \underline{\tau}_O$

- In matrix form,  $M \begin{bmatrix} \dot{\mathbf{v}}_O \\ \dot{\boldsymbol{\omega}} \end{bmatrix} + \begin{bmatrix} \boldsymbol{\omega}^\times & \mathbf{0} \\ \mathbf{v}_O^\times & \boldsymbol{\omega}^\times \end{bmatrix} M \begin{bmatrix} \mathbf{v}_O \\ \boldsymbol{\omega} \end{bmatrix} = \begin{bmatrix} \mathbf{f} \\ \boldsymbol{\tau}_O \end{bmatrix}$ 
  - \*  $\begin{bmatrix} \mathbf{v}_O \\ \boldsymbol{\omega} \end{bmatrix}$  is a generalized velocity and  $\begin{bmatrix} \mathbf{f} \\ \boldsymbol{\tau}_O \end{bmatrix}$  is a generalized force
- If  $O = \bullet$ , we can simplify (where all quantities are relative to the centre of mass):
  - \*  $m\dot{\mathbf{v}} + m\boldsymbol{\omega}^\times \mathbf{v} = \mathbf{f}$
  - \*  $\mathbf{I}\dot{\boldsymbol{\omega}} + \boldsymbol{\omega}^\times \mathbf{I}\boldsymbol{\omega} = \boldsymbol{\tau}$
- Note that in the general case, the equations of motion are coupled; but if we use  $\bullet$ , the rotational equation is uncoupled, making it much easier to solve
- Solving this gives us the angular velocity of the body, but not the orientation; for that we need to use Poisson's equation  $\dot{\mathbf{C}} + \boldsymbol{\omega}^\times \mathbf{C} = \mathbf{0}$ , or for Euler angles  $\boldsymbol{\omega} = \mathbf{S}\dot{\boldsymbol{\theta}}$  (or axis-angle/quaternion)
- Kinetic energy:  $T = \frac{1}{2} \int_R \dot{\mathbf{r}} \cdot \dot{\mathbf{r}} \, dm$ 

$$= \frac{1}{2} \int_R (\mathbf{v}_O - \boldsymbol{\rho} \times \boldsymbol{\omega}) \cdot (\mathbf{v}_O - \boldsymbol{\rho} \times \boldsymbol{\omega}) \, dm$$

$$= \frac{1}{2} \int_R (\mathbf{v}_O - \boldsymbol{\rho} \times \boldsymbol{\omega})^T (\mathbf{v}_O - \boldsymbol{\rho} \times \boldsymbol{\omega}) \, dm$$

$$= \frac{1}{2} m \mathbf{v}_O^T \mathbf{v}_O - \mathbf{v}_O^T \mathbf{c}_O^\times \boldsymbol{\omega} + \frac{1}{2} \boldsymbol{\omega}^T \mathbf{J}_O \boldsymbol{\omega}$$
  - Notice that this has 2 parts: translational, rotational, and a coupling term, which disappears when we use the centre of mass reference frame
  - In matrix form,  $\mathbf{T} = \frac{1}{2} \begin{bmatrix} \mathbf{v}_O \\ \boldsymbol{\omega} \end{bmatrix}^T M \begin{bmatrix} \mathbf{v}_O \\ \boldsymbol{\omega} \end{bmatrix}$
- Note we can expand the inertia matrix as  $\mathbf{I} = \int_R \begin{bmatrix} y^2 + z^2 & -xy & -xz \\ -xy & x^2 + z^2 & -yz \\ -xz & -yz & x^2 + y^2 \end{bmatrix} dm$ 
  - In general  $\mathbf{I}$  will be fully populated; but can we diagonalize it?
  - We know  $\mathbf{I}$  is symmetric positive definite, so it is diagonalizable and the eigenvector matrix is orthogonal
  - There will always be a  $\mathbf{E}$  such that  $\mathbf{E}^{-1} \mathbf{I} \mathbf{E} = \boldsymbol{\Lambda}$ ; but since  $\mathbf{E}^{-1} = \mathbf{E}^T$  (and choose  $\mathbf{E}$  such that  $\det \mathbf{E} = 1$ ), it is a rotation matrix
  - Recall that the inertia matrix transforms as  $\mathbf{C}_{ab} \mathbf{I}_b \mathbf{C}_{ba} = \mathbf{I}_a$ , which has the exact same form as the diagonalization we found
  - Therefore we can always find a reference frame such that  $\mathbf{I}$  is diagonal; this is referred to as the *principal-axis frame*
  - This works even if we don't use  $\bullet$  as our reference point
- In the principal axis frame, with  $\bullet$  as our reference point, the rotational equation reduces to:
  - $I_1 \dot{\omega}_1 - (I_2 - I_3) \omega_2 \omega_3 = \tau_1$
  - $I_2 \dot{\omega}_2 - (I_3 - I_1) \omega_3 \omega_1 = \tau_2$
  - $I_3 \dot{\omega}_3 - (I_1 - I_2) \omega_1 \omega_2 = \tau_3$
  - These are known as *Euler's equations*

## Summary

For a rigid body, let  $\boldsymbol{\rho}$  be the position of a differential mass element relative to  $O$  in a body-fixed frame, then:

- The *first moment of mass/inertia*  $\mathbf{c}_O = \int_R \boldsymbol{\rho} \, dm$ , which is zero if  $O = \ominus$
- The *second moment of mass/inertia matrix*  $\mathbf{J}_O = - \int_R \boldsymbol{\rho}^\times \boldsymbol{\rho}^\times \, dm$ , which is diagonal in the principal axis frame and denoted  $\mathbf{I}$  if  $O = \omin�$

Then the linear and angular momenta are given by

$$\mathbf{p} = m\mathbf{v}_O - \mathbf{c}_O^\times \boldsymbol{\omega}, \quad \mathbf{h}_O = \mathbf{c}_O^\times \mathbf{v}_O + \mathbf{J}_O \boldsymbol{\omega}$$

The equations of motion are given by, in general,

$$\begin{aligned} m\dot{\mathbf{v}}_O - \mathbf{c}_O^\times \dot{\boldsymbol{\omega}} + m\boldsymbol{\omega}^\times \mathbf{v}_O - \boldsymbol{\omega}^\times \mathbf{c}_O^\times \mathbf{v}_O &= \mathbf{f} \\ \mathbf{c}_O^\times \dot{\mathbf{v}}_O + \mathbf{J}_O \dot{\boldsymbol{\omega}} - \mathbf{c}_O^\times \boldsymbol{\omega}^\times \mathbf{v}_O + \boldsymbol{\omega}^\times \mathbf{J}_O \boldsymbol{\omega} &= \boldsymbol{\tau}_O \end{aligned}$$

Using  $O = \omin�$ , this reduces to

$$\begin{aligned} m\dot{\mathbf{v}} + m\boldsymbol{\omega}^\times \mathbf{v} &= \mathbf{f} \\ \mathbf{I}\dot{\boldsymbol{\omega}} + \boldsymbol{\omega}^\times \mathbf{I}\boldsymbol{\omega} &= \boldsymbol{\tau} \end{aligned}$$

And the kinetic energy is given by, in general

$$T = \frac{1}{2} m \mathbf{v}_O^T \mathbf{v}_O - \mathbf{v}_O^T \mathbf{c}_O^\times \boldsymbol{\omega} + \frac{1}{2} \boldsymbol{\omega}^T \mathbf{J}_O \boldsymbol{\omega}$$

where the middle coupling term disappears when using  $O = \omin�$ .