

Lecture 27, Nov 17, 2022

Fin Sizing

- When can we assume a fin to be infinitely long?
- Compare adiabatic tip vs. infinitely long fin equation, taking the ratio we get $\tanh(aL)$
- For $aL = 1$, $\tanh(aL) = 0.762$; for $aL = 5$, $\tanh(aL) = 0.9999$
 - As a rule of thumb if $aL \geq 5$ we can assume the fin is infinitely long
 - A value of 1 gives 76.2% of the total heat transfer from an infinitely long fin, with a lot less material
- $L = \frac{1}{a}$ is typically a reasonable length for a fin
- What about the area of fins?
 - Consider area with fin and area without fin
 - $\dot{Q}_{total} = \dot{Q}_{nofin} + \dot{Q}_{fin}$
 - From the fin efficiency definition $\eta_{fin} = \frac{\dot{Q}_{fin}}{hA_{fin}(T_b - T_\infty)}$
 - $\dot{Q}_{fin} = h\eta_{fin}A_{fin}(T_b - T_\infty)$
 - Putting it all together $\dot{Q}_{total} = h(A_{nofin} + \eta_{fin}A_{fin})(T_b - T_\infty)$
- $(A_{nofin} + \eta_{fin}A_{fin})$ is the *effective area for heat transfer*
 - In terms of thermal resistances $R = \frac{T_b - T_\infty}{\dot{Q}_{total}} = \frac{1}{h(A_{nofin} + \eta_{fin}A_{fin})}$

Transient Heat Conduction (Lumped)

- Simple example: taking a material and immersing it in a fluid with a high temperature difference, resulting in rapid heat transfer
 - What is $T(t)$?
- Recall $\frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} + \frac{\partial^2 T}{\partial z^2} = \frac{1}{\alpha} \frac{\partial T}{\partial t}$
- Lumped capacitance: simple assumption that there is no temperature gradient in the body, i.e. $\frac{\partial^2 T}{\partial x^2} = \frac{\partial^2 T}{\partial y^2} = \frac{\partial^2 T}{\partial z^2} = 0$, the temperature is at uniform temperature everywhere
 - In reality modes of heat transfer include conduction within the material and convection to the surrounding fluid
 - This makes sense only if $R_{cond} \ll R_{conv}$
- Energy balance: Let $\Delta \dot{E}(t) = -\dot{Q}_{conv}(t)$ be the energy change in the body
 - $\Delta \dot{E}(t) = mc_p \frac{dT}{dt} = \rho V c_p \frac{dT}{dt}$
 - $hA(T - T_\infty) = -\rho V c_p \frac{dT}{dt} \implies \frac{dT}{T - T_\infty} = \frac{d(T - T_\infty)}{T - T_\infty} = -\frac{hA}{\rho V c_p} dt$
 - Integrate to get $\ln(T - T_\infty) = -\frac{hA}{\rho V c_p} t + c_1$, apply boundary condition that $T(0) = T_i$
 - $\ln\left(\frac{T - T_\infty}{T_i - T_\infty}\right) = -\frac{hA}{\rho V c_p} t$
 - This compares the “changing driving force” against the “max driving force”
- Define the time constant $\tau = \frac{\rho V c_p}{hA}$ so $\frac{T - T_\infty}{T_i - T_\infty} = e^{-\frac{t}{\tau}}$

Equation

For lumped transient heat conduction,

$$\frac{T - T_\infty}{T_i - T_\infty} = e^{-\frac{t}{\tau}}$$

where $\tau = \frac{\rho V c_p}{hA}$

Validity of the Lumped Capacitance Assumption

- This only makes sense if $R_{conv} \gg R_{cond}$
- Take $\frac{R_{cond}}{R_{conv}} = \frac{\frac{L}{kA}}{\frac{1}{hA}} = \frac{hL}{k}$
- $Bi = \frac{hL_c}{k}$ is the *Biot number*, a dimensionless quantity
 - L_c is a characteristic length in the direction of conduction, from the midpoint to the wall
 - $L_c = \frac{V}{A}$
 - For a sphere $L_c = \frac{r}{3}$, for a cylinder (with length \gg radius) $L_c = \frac{r}{2}$
- Consider the steady state analogue; for $Bi \gg 1$, the temperature drops the sharpest over conduction, so lumped capacitance is not valid; for $Bi \ll 1$, the temperature drops the sharpest over convection, so lumped capacitance is valid
- Typical cutoff is $Bi < 0.1$
- Example: putting steel rod at 300°C into furnace at 1200°C with $h = 100\text{W/m}^2\text{K}$, $D = 0.1\text{m}$, $k = 51.2\text{W/mK}$, $\rho = 7832\text{kg/m}^3$, $c = 541\text{J/kgK}$, how long until the rod temperature reaches 800°C ?
 - First, check validity of lumped capacitance assumption: $Bi = \frac{hL_c}{k} = \frac{100\text{W/m}^2\text{K} \cdot \frac{0.05\text{m}}{2}}{51.2\text{W/mK}} = 0.05 < 0.1$ so the assumption is valid
 - Using lumped capacitance $\frac{T - T_\infty}{T_i - T_\infty} = e^{-\frac{hA}{\rho V c} t} = e^{-\frac{h}{\rho c} \cdot \frac{2}{r} t}$
 - $\ln \frac{800 - 1200}{300 - 1200} = \frac{-2 \cdot 100\text{Wm}^2\text{K}}{7832\text{kg/m}^3 \cdot 541\text{Jkg}\cdot\text{K}} t$
 - $t = 859\text{s}$