Lecture 35, Dec 2, 2022

Models of the Fluid

- To develop any governing equation, we start from physical principles (e.g. conservation of mass, energy), use a suitable model of the fluid, and then derive mathematical equations
 - We have seen the finite control volume and finite system, now we will see the infinitesimal control volume and infinitesimal system
- We take an infinitesimally small control volume and system moving with the flow, such that all properties in this volume are constant

Substantial Derivative

- Consider an infinitesimally small fluid element, at point 1 at $t = t_1$ and travelling to point 2 at $t = t_2$
- Let $\vec{V} = u\hat{i} + v\hat{j} + w\hat{k}$ be the velocity field, where u, v, w are functions of (x, y, z, t)
- Let $\rho = \rho(x, y, z, t)$ be the density • Initially the fluid particle has $\rho_1 = \rho(x_1, y_1, z_1, t_1)$ and velocity $\vec{V_1}$; at time $t = t_2$ the particle has $\rho_2 = \rho(x_2, y_2, z_2, t_2)$ and \vec{V}_2

$$\rho_2 = \rho(x_2, y_2, z_2, t_2) \text{ and } V_2$$
• Taylor expand density around point 1: $\rho_2 = \rho_1 + \frac{\partial \rho}{\partial x}(x_2 - x_1) + \frac{\partial \rho}{\partial y}(y_2 - y_1) + \frac{\partial \rho}{\partial z}(z_2 - z_1) + \frac{\partial \rho}{\partial t}(t_2 - t_1) + \cdots$

$$-\frac{\rho_2 - \rho_1}{t_2 - t_1} = \frac{\partial \rho}{\partial x} \frac{x_2 - x_1}{t_2 - t_1} + \frac{\partial \rho}{\partial y} \frac{y_2 - y_1}{t_2 - t_1} + \frac{\partial \rho}{\partial z} \frac{z_2 - z_1}{t_2 - t_1} + \frac{\partial \rho}{\partial t}$$

- If we take $\lim_{t_2 \to t_1}$, we get $\frac{D\rho}{Dt} = u \frac{\partial \rho}{\partial x} + v \frac{\partial \rho}{\partial y} + w \frac{\partial \rho}{\partial z} + \frac{\partial \rho}{\partial t}$

- * The notation ^{Dρ}/_{Dt} indicates that we're following the same system
 * This is called a *substantial derivative* The substantial derivative ^D/_{Dt} is Lagrangian, while the right hand side with all the partial derivatives is Eulerian

Definition

The substantial derivative operator is defined as

$$\frac{\mathrm{D}}{\mathrm{D}t} = \frac{\partial}{\partial t} + u\frac{\partial}{\partial x} + v\frac{\partial}{\partial y} + w\frac{\partial}{\partial z} = \frac{\partial}{\partial t} + \vec{V}\cdot\vec{\nabla}$$

where $\vec{V} \cdot \vec{\nabla}$ is the convective derivative

- The substantial derivative is made of the local derivative, time rate of change at a fixed point due to unsteady fluctuations, and the *convective derivative* $u\frac{\partial}{\partial x} + v\frac{\partial}{\partial y} + w\frac{\partial}{\partial z}$, which is the time rate of change as a result of the movement of fluid
- The substantial derivative is a total derivative with respect to time

Divergence of Velocity

- $\vec{\nabla} \cdot \vec{V} = \frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z}$ can be interpreted as the time rate of change of the volume of an infinitesimal moving system per unit volume
- As the system is moving, its volume is continuously changing
- $\vec{\nabla} \cdot \vec{V} = \frac{1}{\delta\Omega} \frac{D(\delta\Omega)}{Dt}$ Note that if we have an incompressible flow, constant density means $\delta\Omega_{sys}$ must be constant, so $\frac{\mathbf{D}(\delta\Omega)}{\mathbf{D}t} = 0, \text{ this means } \vec{\nabla} \cdot \vec{V} = 0$

General Continuity Equation

- We're starting with mass conservation, move through the 4 different models to get the continuity equation
- Recall for the finite control volume we have $\frac{\mathrm{d}}{\mathrm{d}t} \iiint_{\Omega} \rho \,\mathrm{d}\Omega + \oint_{S} \rho \vec{V} \cdot \mathrm{d}\vec{S} = 0$ which we derived from RTT
- For the finite system, we follow the system and take $\frac{Dm_{sys}}{Dt}$, which by mass conservation should be 0, which gives us $\frac{D}{Dt} \iiint_{\Omega} \rho \, d\Omega = 0$ • For the infinitesimal control volume:
- - Along a side we have the flow rate as $\left(\rho u + \frac{\partial(\rho u)}{\partial x} dx\right) dy dz$ - The net mass flow rate out of the control volume is $\left(\frac{\partial(\rho u)}{\partial x} + \frac{\partial(\rho u)}{\partial y} + \frac{\partial(\rho w)}{\partial z}\right) dx dy dz$
 - This is equal to $-\frac{\partial}{\partial t} dx dy dz$ due to conservation of mass $\frac{\partial \rho}{\partial t} = \frac{\vec{x}}{\partial t} (\vec{x}) = 0$

- Combined this gives us
$$\frac{\partial \rho}{\partial t} + \vec{\nabla} \cdot (\rho \vec{V}) = 0$$

ow consider the infinitesimal system:

• Now consider the infinitesimal system: $-\frac{D(\delta m)}{Dt} = 0 \text{ where } \delta m = \rho \delta \Omega$

$$-\frac{\overline{D(\delta m)}}{Dt} = \frac{D}{Dt}(\rho\delta\Omega) = \rho \frac{D}{Dt}(\Delta\Omega) + \delta\Omega \frac{D\rho}{Dt} = 0$$
$$-\frac{D\rho}{Dt} + \rho \left(\frac{1}{\delta\Omega} \frac{D(\Delta\Omega)}{Dt}\right) = 0$$
$$-\text{ This gives us } \frac{D\rho}{Dt} + \rho \vec{\nabla} \cdot \vec{V} = 0$$

• This gave us 4 general forms of the continuity equation:

1.
$$\frac{\mathrm{d}}{\mathrm{d}t} \iiint_{\Omega} \rho \,\mathrm{d}\Omega + \oiint_{S} \rho \vec{V} \cdot \mathrm{d}\vec{S} = 0$$

2.
$$\frac{\mathrm{D}}{\mathrm{D}t} \iiint_{\Omega} \rho \,\mathrm{d}\Omega = 0$$

3.
$$\frac{\partial \rho}{\partial t} + \vec{\nabla} \cdot (\rho \vec{V}) = 0$$

4.
$$\frac{\mathrm{D}\rho}{\mathrm{D}t} + \rho \vec{\nabla} \cdot \vec{V} = 0$$

• For a steady flow, $\frac{\partial}{\partial t} = 0$ but $\frac{D}{Dt} \neq 0$, so we have:

- 1. $\iint_{S} \rho \vec{V} \cdot d\vec{S} = 0$ 2. $\iint_{Dt} \iiint_{\Omega} \rho d\Omega = 0$ 3. $\vec{\nabla} \cdot (\rho \vec{V}) = 0$
- 4. $\frac{D\rho}{Dt} + \rho \vec{\nabla} \cdot \vec{V} = 0$ For an compressible flow, ρ is constant and so:

1.
$$\oint_{S} \rho \vec{V} \cdot d\vec{S} = 0$$

2.
$$\frac{D}{Dt} \Omega = 0$$

3.
$$\vec{\nabla} \cdot \vec{V} = 0$$

- Note 3 and 4 became the same

• All these forms are mathematically equivalent